

A Computational Study of Solar Air Heater for Its Performance Enhancement by Numerical Approach Using Standard CFD Tools

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Abstract

In the following report, the main objective is to calculate the efficiency of the solar air heater, mean exit temperature of air using CFD (Computational Fluid Dynamics) simulation. CFD software promises to be a futuristic field of application especially in the field of thermal engineering. The analysis is performed on a two-dimensional built solar air heater. The geometry modelling is done in "Gambit" software followed by its analysis for mean exit temperature of air in "Fluent" (ANSYS). The solar air heater is modelled using both rectangular and circular shaped fins placed in an alternate fashion to give high turbulence effect to air flow. Few elementary calculations are also done in order to find the net heat flux and hydraulic diameter of the duct.

Keyword: Computational fluid dynamics (CFD), gambit, solar air heater

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INTRODUCTION

A two-dimensional solar air heater is to be constructed as per the dimensions mentioned below in the figure. The heater consists of three main regions namely the entry region, test region and exit region. Air is made to pass through the heater at the inlet of the entry region. The air is heated by a constant heat flux acting over the boundary enclosed by the fins in the test region (Figures 1–6).^[1–3]

Calculate the following:

- 1) The Heat Flux acting by analytical solution
- 2) For the six mass flow rates of air mentioned in the tables, find the exit mean temperature, heat content of exit air and efficiency of the heater through CFD simulation.

Sr. No	Mass Flow Rate(kg/s)
1.	0.014
2.	0.02
3.	0.025
4.	0.03

- 3) Also plot the graphs of the following:
^[2–4]

- Exit mean temperature vs mass flow rate.
- Heat content of exit air vs mass flow rate.
- Efficiency of the heater versus mass flow rate.

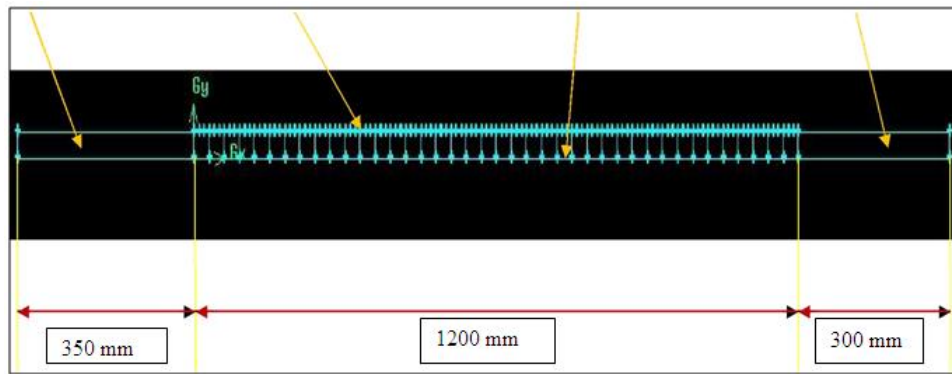


Fig. 1: Uniform UDL Meshing of Top Layer of Plate.

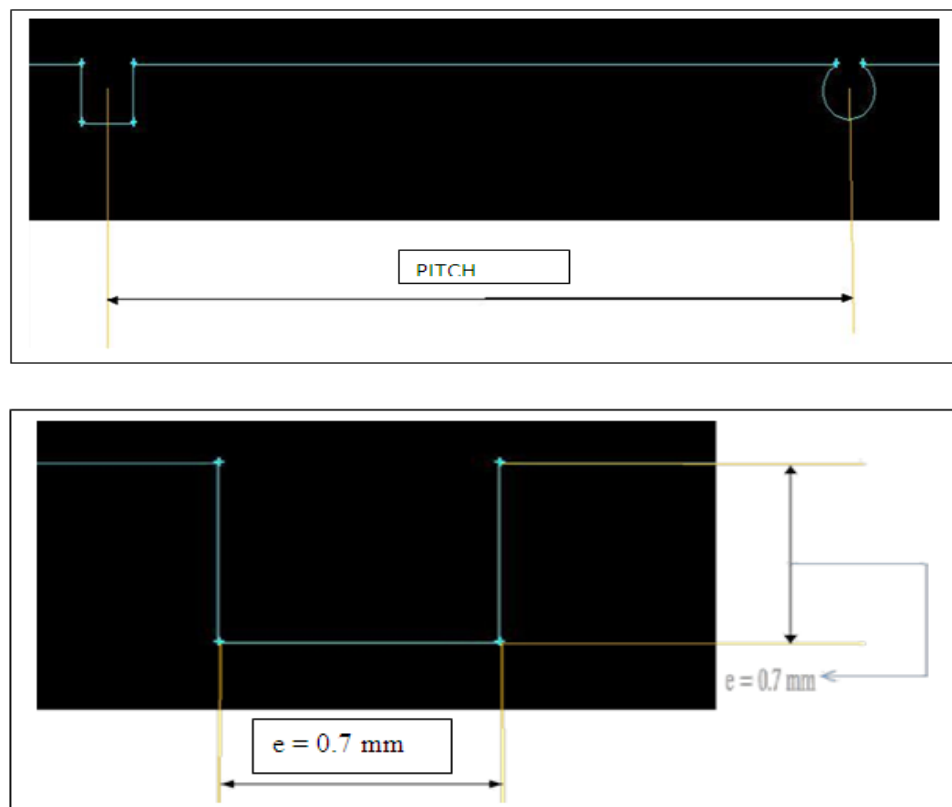


Fig. 2: Internal Layer of Solar Heater with Counter to Increase the Turbulence.

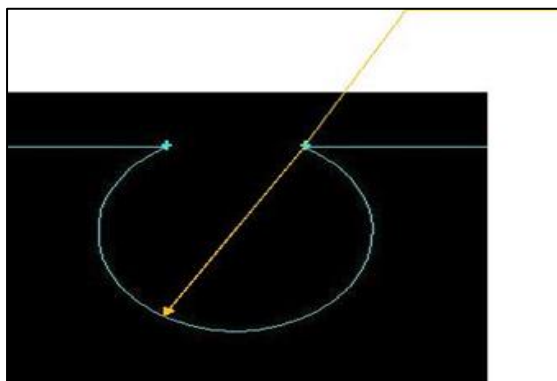


Fig. 3: Internal Layer of Solar Heater with Circular Counter to Increase the Turbulence.

MODELING IN GAMBIT SOFTWARE

General Commands Used ^[4-6]

Geometry Command Button
 Vertex Command Button
 Edge Command Button
 Face Command Button
 Connect vertices/edges/faces
 Delete vertices/edges/faces/meshes

Mesh Command Button

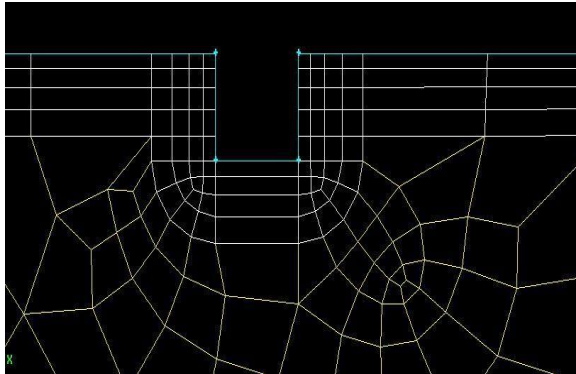


Fig. 4: Mesh Around Square Fin.

Boundary Layer Command button

First Row, $a = 0.1$

Growth Factor, $b/a = 1.2$

Rows = 4

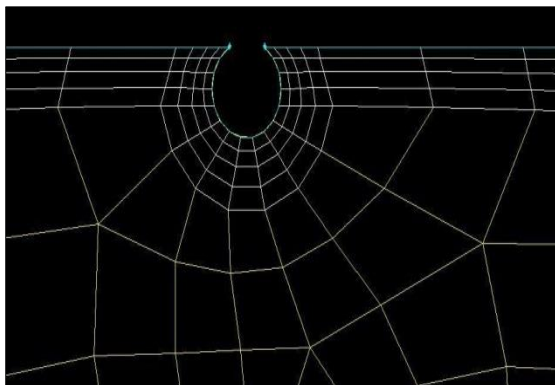


Fig. 5: Mesh Around Circular Fin.

Mesh Edges

Ratio = 1

Interval size = 15

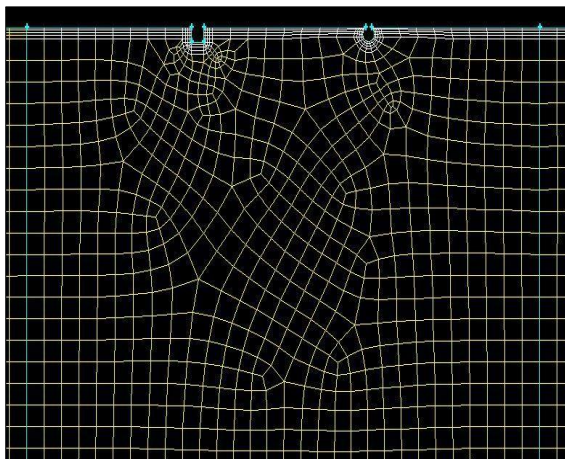


Fig. 6: Meshing the Whole Plane.

Mesh Faces

Elements: Quad

Type: Pave

Interval size: 1

Zones Command Button

Specify boundary types

Name	Type
Heat In	Wall
Mass In	Mass Flow Inlet
Mass Out	Pressure Outlet

Specify Continuum Types

Region	Type
Inlet	Fluid
Exit	Fluid
Test	Fluid
Absorber plate	Solid

ANALYSING IN FLUENT

The following analysis is done on the case having a mass flow rate of 0.014 kg/s.

Models

Energy – ON

Viscous Model- k-omega, SST, low Recorrections

Materials

Fluid (air)

Solid (Aluminium)

Boundary Conditions

Zone: Heat in (Type-Wall)

Heat Flux = 1000 W

Zone: Mass in (Type-Mass flow inlet)

Mass flow rate = 0.0933 kg/s

Direction Specification Method: Normal to Boundary

Turbulence Specification Method: Intensity & Hydraulic diameter Turbulent Intensity = 3%,

Hydraulic diameter = 0.05 m

Total temperature = 307 K

Solution Methods

Scheme-Simple

Gradient-Least squares cell based

Pressure-Standard

upwind

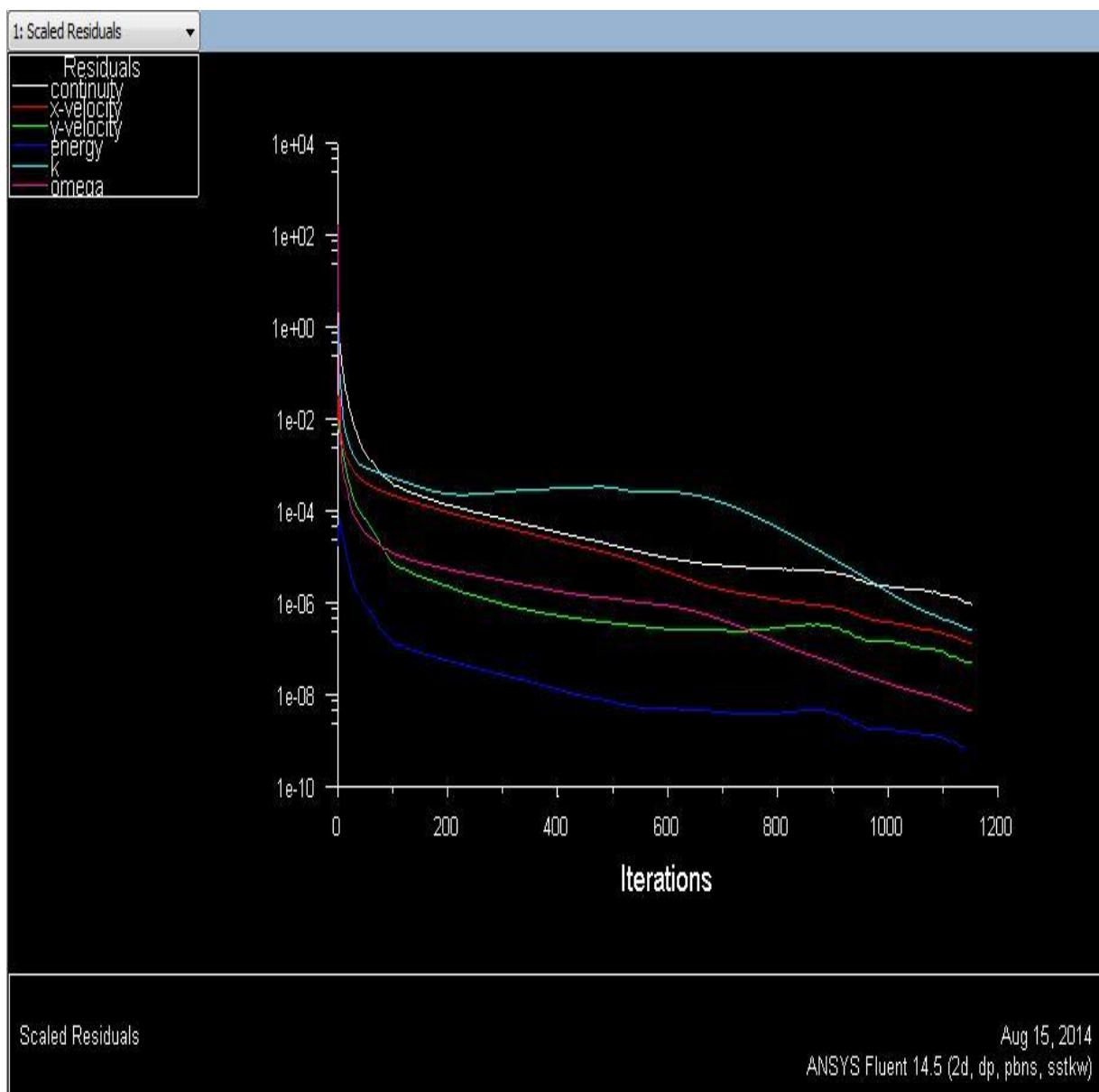
Momentum-Second order upwind

Specific Dissipation Rate – Second order
upwind (Figures 7, 8).

Turbulent Kinetic Energy – Second order

Residual Monitors

Residual	Absolute Convergence Criteria
Continuity	1e-06
x-velocity	1e-06
y-velocity	1e-06
Energy	1e-06

Number of Iterations: 100000

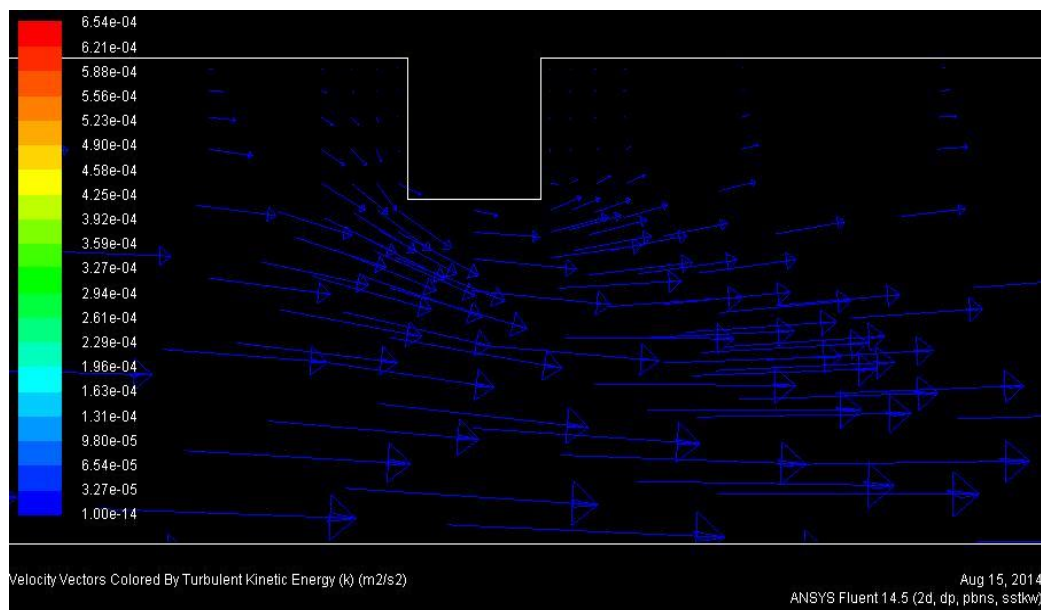


Fig. 7: Laminar Compressed Flow across the Counter.

CALCULATIONS:

Given data:

Inlet temperature = 307 K

$C_p = 1005 \text{ J/kgK}$

$m = 0.0933 \text{ kg/s}$

$Q_{in} = 1000 \text{ f or } 1.2$

$1.0 = 1200 \text{ W}$

$$\begin{aligned} \text{Exit heat energy, } Q_u &= m \times C_p \times (T_o - T_i) \\ &= 0.014 \times 1005 \times (319.96 - 307) \text{ W} \\ &= 182.34 \text{ W} \end{aligned}$$

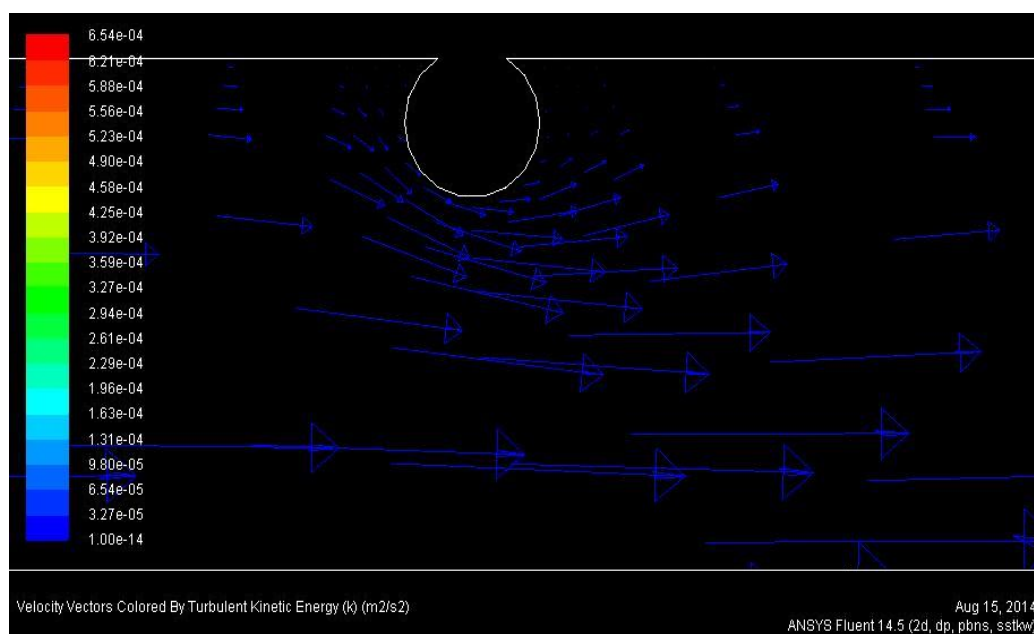
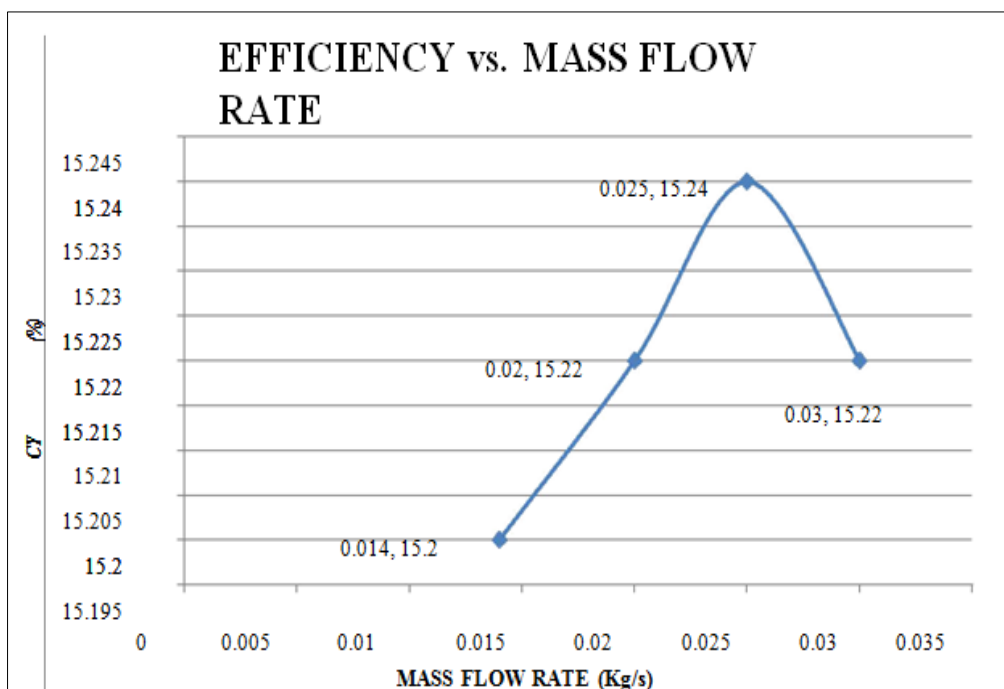
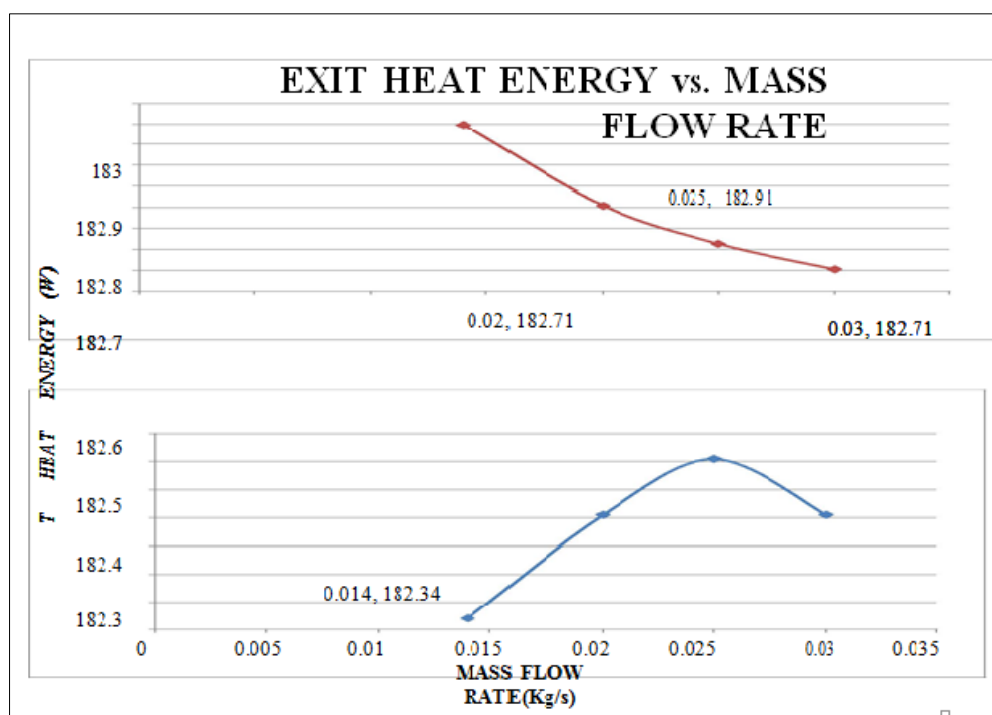


Fig. 8: Flow Across Circular Fin.



Graph 1: Representation of Efficiency Versus Mass Flow Rate.



Graph 2: Representation of Exit Heat Energy Versus Mass Flow Rate.

Tabulation of Results

Mass Flow Rate (kg/s)	Outlet Temp. (K)	Exit Heat Energy (W)	Efficiency of Heater (%)
0.014	319.96	182.34	15.20
0.02	316.09	182.71	15.22
0.025	314.28	182.91	15.24
0.03	313.06	182.71	15.22

CONCLUSION

As per the given problem statement that is how to increase the heat transfer in a two dimensional solar air heater under different mass flow rate ranging from 0.014 to 0.03 kg/s and also by varying the material properties to get these solution listed as below:

1. The Exit temperature decreases with increase in mass flow rate ranging from (0.014 to 0.03 kg/s) and also with varying material Properties.
2. Exit Heat Energy of air increases with increase in mass flow rate and with the more Surface area Contact between the Air and the heated surface
3. Hence the efficiency of the solar air heater increases with increase in mass flow rate and also with varying material with different attributes.

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